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(54) Abstract Title

Drill bit

(57) A rolling cone drill bit has a bit body having at least one leg, including a journal segment and a shirttail 14. The shirttail 14 has a leg backface 18. A rolling cone cutter 20 has a cone backface 26. The leg backface 18 is canted relative to the cone bearing axis (angle alpha). A seal 40 is positioned in the interface between the leg backface 18 and the cone backface 26. In a further embodiment the seal 40 has an operating axis that is canted with respect to the bearing axis (angle beta).

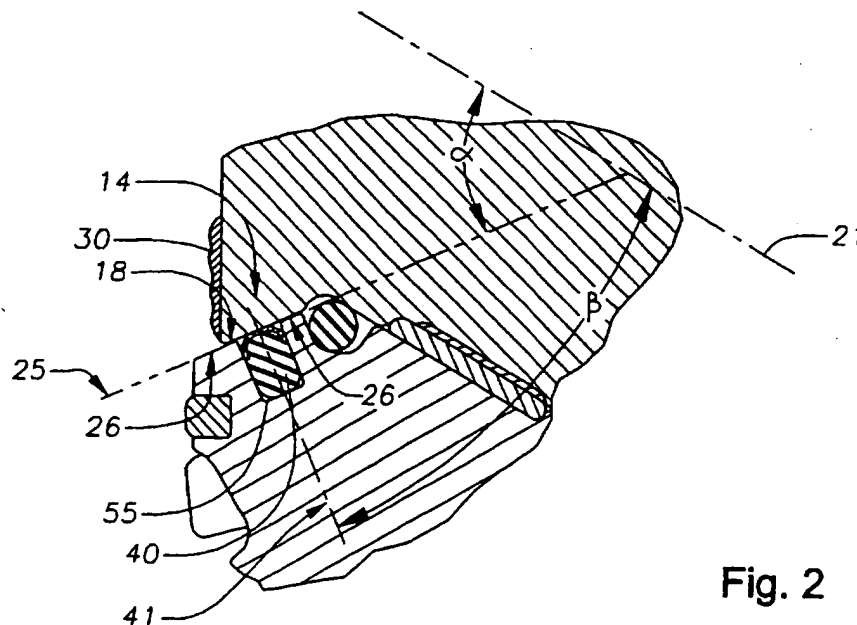
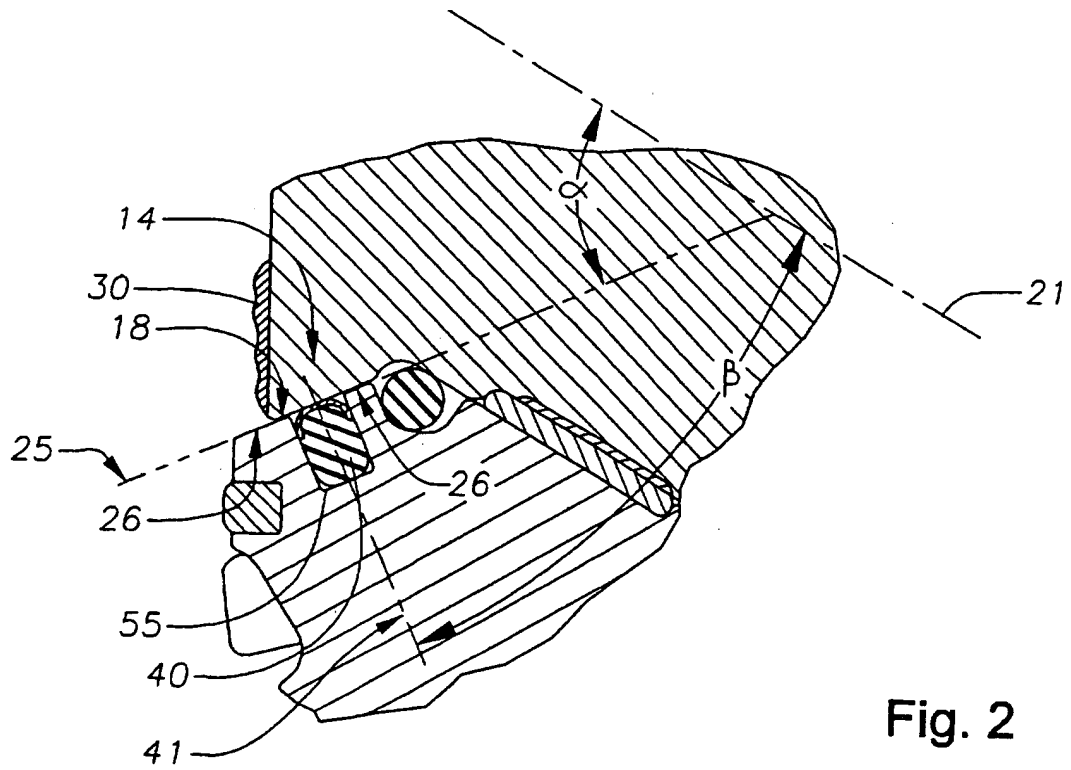
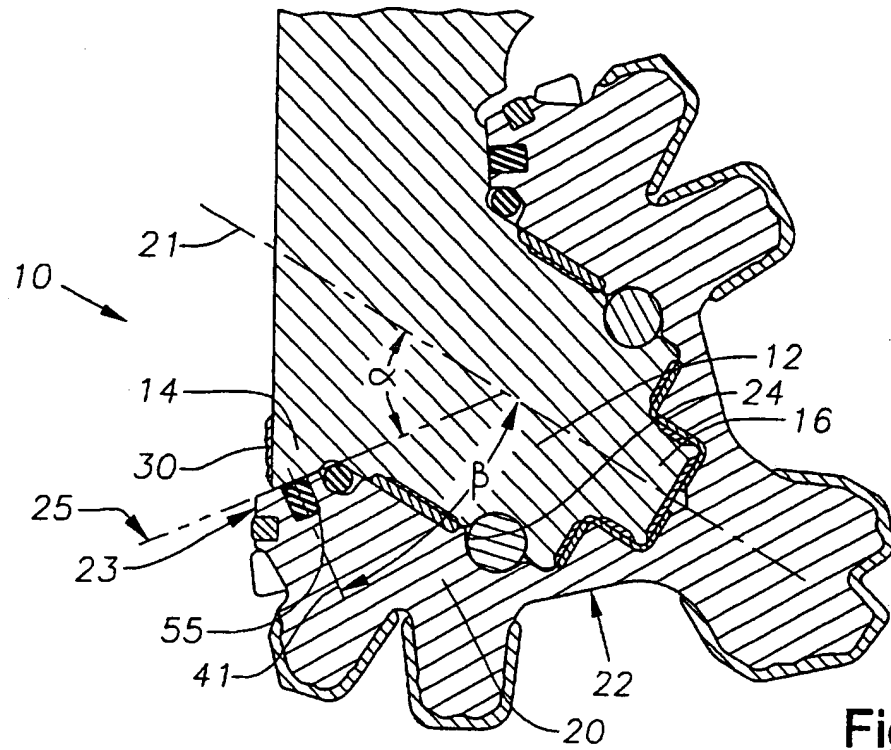
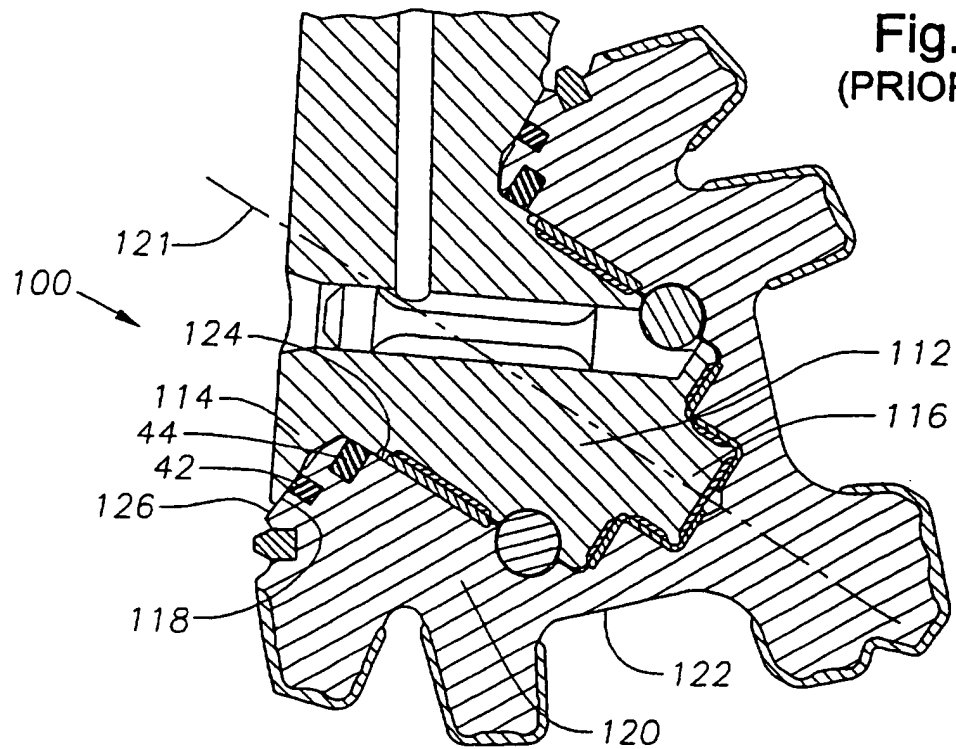
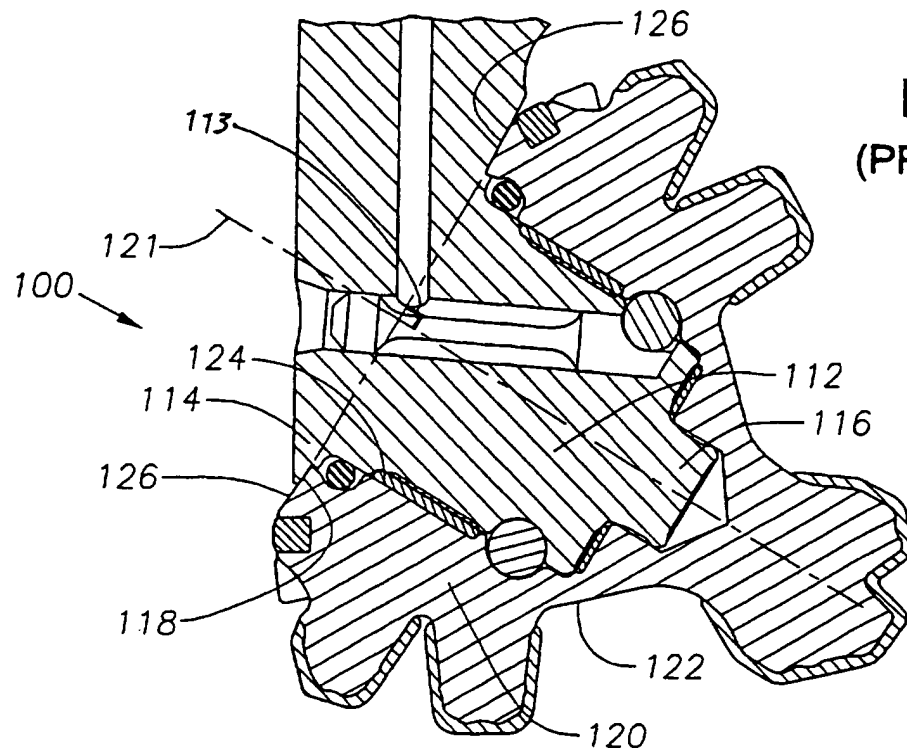


Fig. 2

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4/4

Fig. 5

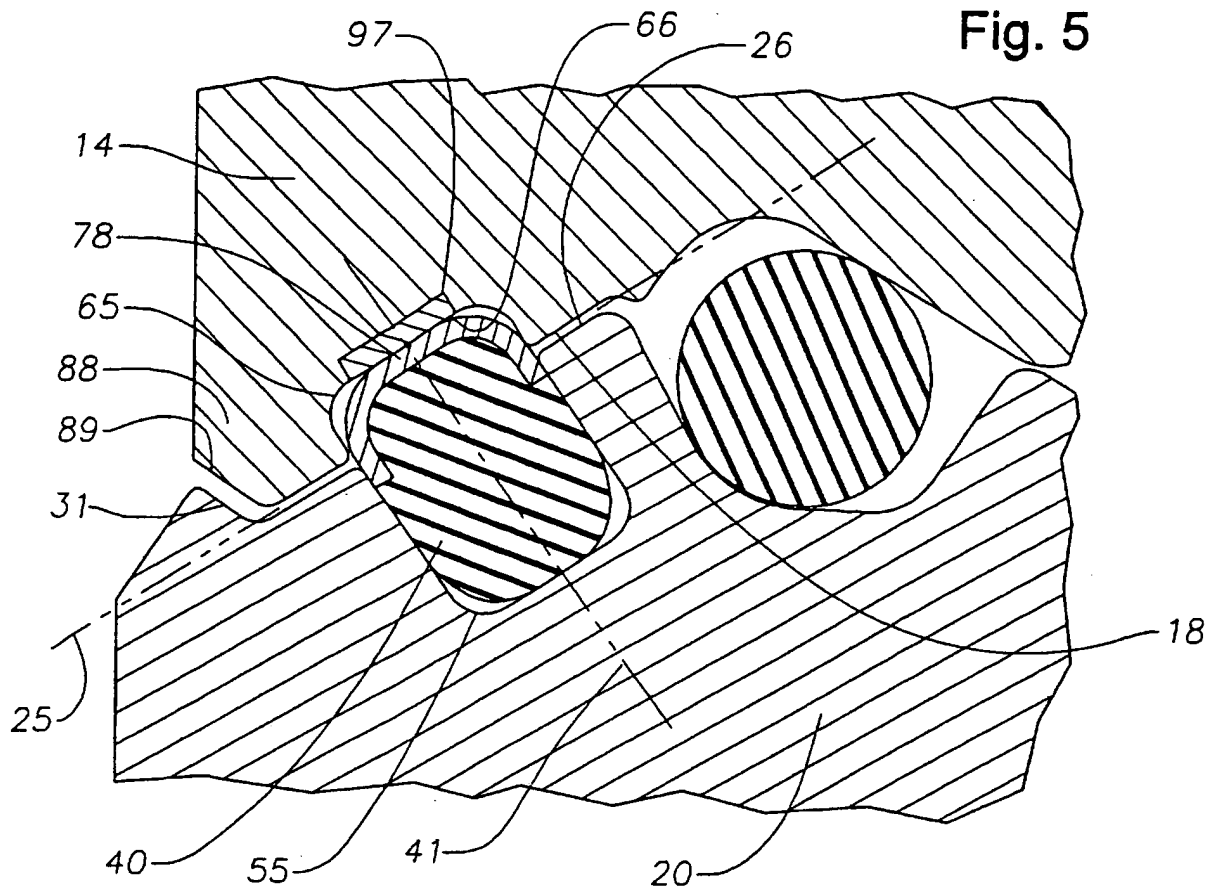
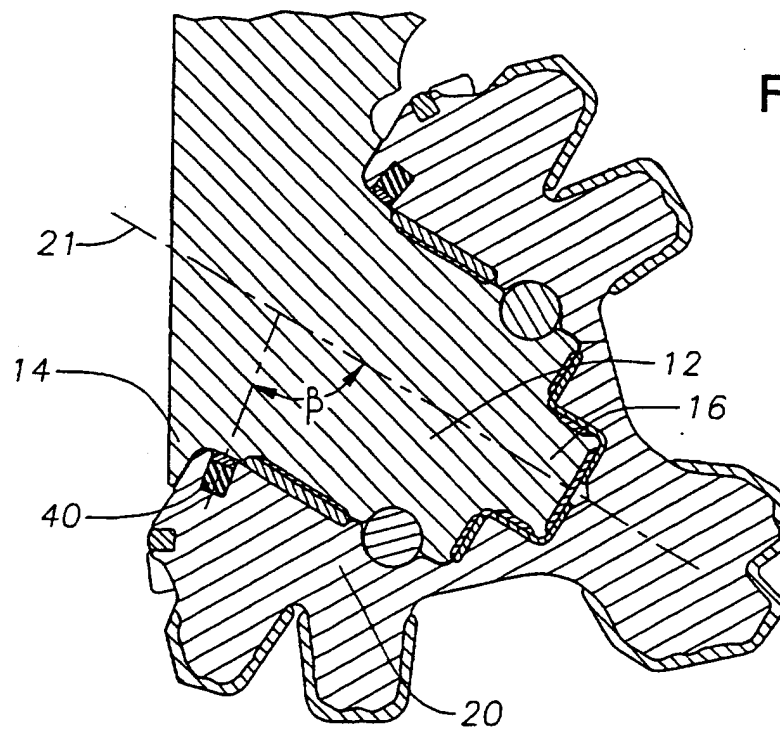


Fig. 6



DRILL BIT

The present invention relates to a drill bit.

5 The present invention relates particularly to sealed bearing earth boring drill bits, such as rotary cone rock bits. More particularly, the present invention relates to the seals that are used to seal the bearing area between the bit leg and the rolling cone. Still more particularly,
10 the present invention relates to a seal that is canted or angled with respect to the bearing axis and also to a canted leg/cone interface.

 An earth-boring drill bit is typically mounted on the
15 lower end of a drill string and is rotated by rotating the drill string at the surface or by actuation of downhole motors or turbines, or by both methods. With weight applied to the drill string, the rotating drill bit engages the earthen formation and proceeds to form a borehole along
20 a predetermined path toward a target zone. A typical earth-boring bit includes one or more rotatable cutters, or cones. The cutters roll and slide upon the bottom of the borehole as the bit is rotated, thereby engaging and disintegrating the formation material in the bit's path.
25 The rotatable cutters may be described as generally conical in shape and are therefore sometimes referred to as rolling cones.

 Bits designed to work with rolling cones typically
30 include a bit body with a plurality of journal segment legs. The rolling cones are mounted on bearing pin shafts

that extend downwardly and inwardly from the journal segment legs. The borehole is formed as the gouging and scraping or crushing and chipping action of the rotary cones removes chips of formation material. The chips are
5 carried upward and out of the borehole by drilling fluid that is pumped downwardly through the drill pipe and out of the bit. The drilling fluid carries the chips and cuttings in a slurry as it flows up and out of the borehole.

10 In a typical bit, the earth-disintegrating action of the rolling cone cutters is enhanced by providing the cones with a plurality of cutting elements. Cutting elements are generally of two types: inserts formed of a very hard material, such as tungsten carbide, that are press fit into
15 undersized apertures in the cone surface; or teeth that are milled, cast or otherwise integrally formed from the material of the rolling cone.

The cost of drilling a borehole is proportional to the
20 length of time it takes to drill to the desired depth and location. The time required to drill the well, in turn, is greatly affected by the number of times the drill bit must be changed in order to reach the targeted formation. This is the case because each time the bit is changed, the
25 entire string of drill pipe, which may be miles or kilometres long, must be retrieved from the borehole, section by section. Once the drill string has been retrieved and the new bit installed, the bit must be lowered to the bottom of the borehole on the drill string,
30 which again must be constructed section by section. This process, known as a "trip" of the drill string, requires

considerable time, effort and expense. Accordingly, it is always desirable to employ drill bits that will drill faster and longer without failing.

5 The length of time that a drill bit can be employed before it must be replaced depends on many factors, not the least of which is its ability to resist the wear associated with drilling. All of the components of a bit are subjected to severe wear, as a result of frictional contact
10 with the formation and the drilling fluid. While wear can damage any surface of the bit, the bearing surfaces of the bit, namely those between each journal segment and its associated cone, are particularly vulnerable. If grit or other particles were to enter the annular space between the
15 cone and the journal while the bit was rotating, the bearing surfaces would be quickly destroyed, rendering the bit unusable.

For this reason, a great deal of attention has been
20 given to providing a sealing system that prevents the ingress of particles to the bearing surface. It is desirable to provide a sealing system that does not take up too much of the available space on the bit leg, yet provides a good seal. It is further desired to provide a
25 bit that has superior wear resistance and can accommodate additional wear resisting features.

According to a first aspect of the present invention, there is provided an earth-boring bit for drilling a
30 borehole, the bit comprising: a bit body, said body having at least one leg having a journal segment extending

inwardly and downwardly therefrom, said leg including a
shirttail, said shirttail having an outer surface and a leg
backface; and, at least one rolling cone cutter rotatably
mounted on said bit body for rotation about a bearing axis
5 and having a generally conical outer surface and a cone
backface, said cone backface defining an interface with
said leg backface; wherein said leg backface is canted.

According to a second aspect of the present invention,
10 there is provided an earth-boring bit for drilling a
borehole, the bit comprising: a bit body, said body having
at least one leg having a journal segment extending
inwardly and downwardly therefrom, said leg including a
shirttail, said shirttail having an outer surface and a
15 canted leg backface; at least one rolling cone cutter
rotatably mounted on said bit body and having a generally
conical outer surface and a cone backface, said backface
defining an interface with said shirttail inner surface;
and, a seal positioned in said interface.

20

According to a third aspect of the present invention,
there is provided an earth-boring bit for drilling a
borehole, the bit comprising: a bit body, said body having
at least one leg having a journal segment extending
25 inwardly and downwardly therefrom, said leg including a
shirttail, said shirttail having an outer surface and a leg
backface; at least one rolling cone cutter rotatably
mounted on said bit body and having a bearing axis, a
generally conical outer surface, and a backface, said
30 backface defining an interface with said shirttail inner
surface; and, a canted seal positioned in said interface,

said seal having an operating axis that is canted with respect to said bearing axis.

In the preferred embodiment, the present invention
5 provides a sealing system that prevents the ingress of particles to the bearing surface and does not take up too much of the space needed for the bearings. The bit can have superior wear resistance and can accommodate additional wear resisting features. In one preferred
10 embodiment, the bit includes a canted bearing seal whose operating axis is neither parallel nor perpendicular to the bearing axis. The canted bearing seal can be used alone or in combination with additional canted or conventional seals. In another embodiment, the present invention
15 includes a bit having a canted interface between the bit leg and the cone backface. In still another embodiment, the present invention includes a bearing seal that is mounted in a groove in the cone and a corresponding cut-out in the leg, so that there is no straight-line path past the
20 seal.

One embodiment of the present invention comprises a bit that comprises a bit body having at least one leg having a journal segment extending inwardly and downwardly
25 therefrom and at least one rolling cone cutter rotatably mounted on the bit body and having a generally conical outer surface and a cone backface. The leg includes a shirttail, and the shirttail has an outer surface and a leg backface that defines an interface with said cone backface
30 and the leg backface is canted.

In another embodiment, the present bit comprises a bit body and at least one rolling cone cutter rotatably mounted on said bit body. The body has at least one leg having a journal segment extending inwardly and downwardly
5 therefrom, the leg including a shirrtail, and the shirrtail having an outer surface and a canted leg backface. The rolling cone cutter has a generally conical outer surface and a cone backface, and the backface defines an interface with the shirrtail inner surface and a seal is positioned
10 in said interface. This embodiment can be used with a recessed or non-recessed shirrtail and a conventional or a non-positive seal.

Still another embodiment comprises a bit body, and at
15 least one rolling cone cutter rotatably mounted on said bit body. The body has at least one leg having a journal segment extending inwardly and downwardly therefrom, said leg including a shirrtail, said shirrtail having an outer surfaces and a leg backface. The rolling cone cutter has a
20 bearing axis, a generally conical outer surface, and a cone backface. The cone backface defines an interface with said shirrtail inner surface and a canted seal is positioned in the interface. The seal has an operating axis that is canted with respect to said bearing axis.

25

Embodiments of the present invention will now be described by way of example with reference to the accompanying drawings, in which:

30 Figure 1 is a cross-sectional view of an example of a bit leg and associated cone constructed in accordance with

the present invention, wherein the leg backface is canted, the seal is canted, and the seal is received in a groove in the cone;

5 Figure 2 is an enlarged view of the seal of Figure 1;

Figures 3A-B are cross-sectional views of prior art bits, wherein the cone backface is not canted, the seals are not canted, and the shirttail is not recessed;

10

Figure 4 is a cross-sectional view of an example of a bit leg and associated cone constructed in accordance with a preferred embodiment, wherein the leg backface is stepped, the seal is canted, and the seal is semi-
15 encapsulated;

Figure 5 is an enlarged cross-sectional view of a second alternative embodiment of a seal between a bit leg and cone, wherein the leg backface is canted and the seal
20 is semi-encapsulated and comprises a fabric seal; and,

Figure 6 is cross-sectional view of an example of a bit leg and associated cone constructed in accordance with an alternative embodiment of the invention, wherein the
25 seal is received in a groove in the cone and is canted so as to be nearly perpendicular to the bearing axis of the cone.

Referring initially to Figures 1 and 2, a drill bit
30 leg 10 includes a journal segment 12, a shirttail 14 and a bearing end 16. Shirttail 14 includes a leg backface

surface 18. The corresponding cone 20 mounted on journal segment 12 includes a generally conical outer surface 22, a heel surface 23, a stepped inner journal surface 24, and a cone backface 26. A plurality of journal and/or ball
5 bearings 30, 32, 34 are provided in the annular recess 13 between cone 20 and journal segment 12 and allow cone 20 to rotate about the bearing axis 21, as is conventional in the art. It is these bearing components that are protected by the preferred embodiment of the present invention.

10

Specifically, according to the embodiment shown in Figures 1 and 2, a canted leg backface 18 is used in conjunction with a canted seal ring 40. By "canted" it is meant that a line 25 along at least a substantial portion
15 of the backface, as drawn in Figure 1, is not normal to the bearing axis 21. In particular, this is not intended to cover that portion of the backface adjacent to the journal that forms a transition from the journal segment to the leg backface. Conversely, the term "non-canted" is defined to
20 mean that line 25 as drawn in Figure 1 normal to the bearing axis 21. In this embodiment, canted backfaces 18 and 26 each form a straight line as drawn in cross-section and each backface 18,26 is frustoconical.

25 Still referring to Figures 1 and 2, backface line 25 defines an angle α with respect to bearing axis 21. In a preferred embodiment, angle α is between 35 and 85 degrees, with a preferred range being between 45 and 75 degrees. A particularly preferred angle α is approximately 55-65
30 degrees, and most particularly 60 degrees. This modification of the leg backface 18 and shirrtail 14

results in a thicker, more robust shirrtail that has sufficient depth to accommodate a seal such as that shown at 40. While the preferred embodiments include a backface canted at an angle α that is less than 90 degrees, it will
5 be understood that α could alternatively be greater than 90 degrees. This embodiment is not preferred, however, because it reduces the thickness of the shirrtail and the amount of space available for receiving a seal. In the preferred embodiment, the outer diameter of the shirrtail
10 14 is at least 25% and preferably about 40% greater than the diameter of the cylindrical journal segment 12.

Referring briefly to Figure 3A for comparison, a known bit 100 includes a journal segment 112, a shirrtail 114 and
15 a bearing end 116. Shirrtail 114 includes a leg backface 118. Similarly, cone 120 includes a generally conical outer surface 122, a stepped inner journal surface 124, and a cone backface 126. In contrast to the bit shown in Figure 1, however, leg backface 118 lies in a plane that is
20 normal to bearing axis 121 as shown at 113. This in turn means that shirrtail 114 is thinner and of narrow overall diameter and there is less room for a seal to be positioned thereon. Either the seal must be omitted, as shown in Figure 3A, or space must be made to accommodate it. As
25 shown in Figure 3B, seals 42 and 44 can be accommodated by decreasing the diameter of the journal segment 112 and cone inner surface relative to the diameter of cone outer surface 122. This has the undesired effect of weakening journal segment 112. By canting the leg backface 18 as
30 shown in Figures 1 and 2, this undesired weakening can be

avoided while still allowing placement of one or more seals
40.

Referring again to Figures 1 and 2, it can be seen
5 that seal 40 is also canted with respect to the bearing
axis 21 and is received in a correspondingly canted groove
55. In this embodiment, groove 55 is deep enough and wide
enough to receive the seal 40 with an intended degree of
compression of seal 40. According to a preferred
10 embodiment, the operating axis 41 of seal 40 defines an
angle β with respect to the bearing axis 21, where β is
preferably between 10 and 85 and more preferably either
between 15 and 45 degrees or between 70 and 85 degrees.
The term "operating axis" relates to a cross section of the
15 seal and is defined herein to mean the line 41 (as drawn)
along which the seal is designed to be compressed during
operation of the bit. The operating axis 41 and the
backface line 25 need not be normal. Likewise, it is not
necessary that the seal axis 41 be either parallel or
20 perpendicular to the bearing axis 21.

In Figures 1 and 2, the bit of the present invention
can have a non-recessed shirrtail 14, such as is known in
the art and described below. As further discussed in
25 detail below, the bit of the present invention can with
equal advantage have a recessed shirrtail. Similarly,
although the bits illustrated herein have cone backfaces
that echo the configuration of the leg backfaces, the
present invention does not require that the cone faces be
30 canted to the same degree as the leg backfaces, or that
they be canted at all.

Referring now to Figure 4, in a more preferred embodiment, the cone backface 26 and corresponding leg backface 18 each include a change in axial position across a radius of the cone. In this case, however, each backface is "stepped", which means that it comprises at least two non-canted portions a,b, which are axially offset from each other along the bearing axis 21. In Figure 4, seal 40 is again canted with respect to the bearing axis 21 by an angle β , as described above. In this embodiment, backface line 25' is defined by drawing a line between the outer edge of portion a to the inner edge of portion b. As described in the following paragraph, the seal of Figure 4 is preferably semi-encapsulated.

Referring now to Figure 5, a preferred configuration for seal 40 comprises an elongate fabric seal that is received jointly in an annular groove 55 in cone 20 and a corresponding annular cut-out 65 in leg shirttail 14. Groove 55 is preferably aligned with the operating axis 41 of seal 40, and is therefore normal to the cone backface 26. In instances where seal 40 is canted with respect to backface line 25, however, groove 55 is preferably aligned with seal 40 and is not normal to backface 26. Cut-out 65 ensures that no straight-line path exists by which particles can cross seal 40. Like groove 55, the sealing surface 66 of cutout 65 is preferably normal to the seal axis 41. In one preferred embodiment, the sealing surface 66 of cut-out 65 includes a wear resistant insert 97. The seal configuration illustrated in Figure 5 can be applied to advantage to the various embodiments of the present

invention, such that the portion of surface 18 against which seal 40 bears can be recessed and/or include a wear resistant insert or inlay or ring 97.

5 For ease of reference, a seal that is received partially in the cone and partially in the shirrtail in the manner of Figure 5 is referred to herein as a "semi-encapsulated" seal. Semi-encapsulated seals can be canted or not canted with respect to either the leg backface 18 or
10 the bearing axis 21.

The embodiment of Figure 5 also includes a recessed shirrtail 88, wherein the lower edge of shirrtail 88 is received in and protected by the cone 20. As shown, the
15 leading edge of the shirrtail is formed with a flat surface 89 and the cone 20 includes an annular lip 31 surrounding the cone backface surface 26. As a result, shirrtail 14 is protected on its leading edge from contact with the borehole wall, thereby reducing its susceptibility to
20 abrasive wear and fracture. Terminating the shirrtail in an obtuse angle shown in Figure 5 instead of the relatively sharp lower edge shown in Figure 2 significantly reduces breakage of the shirrtail and prolongs the useful life of the bit. The recessed shirrtail also protects the gap
25 between the cone and the leg, preventing debris from entering the gap or air groove and contaminating the seal. Conventional shirrtails tend to erode from their tips and expose components such as seals and bearing elements. By relieving the shirrtail in the manner described herein, the
30 cone itself will protect the shirrtail. As stated above,

the bit of the present invention may or may not have a recessed shirrtail.

Referring finally to Figure 6, an alternative
5 embodiment of the present invention includes a seal 40 that is canted so that the angle β between its operating axis 41 and the bearing axis 21 is in the range of 45 to 85 degrees and more preferably 75 to 85 degrees. As with previously described embodiments, this embodiment provides the
10 advantage of a seal that has a reduced effect on the amount of available bearing space and has both radial and axial activation vectors.

The canted backface disclosed herein provides various
15 advantages. For example, a canted backface rotary cone rock bit can be configured to have a thicker leg shirrtail 14 than conventional rotary cone rock bits. The thicker shirrtail is stronger and, with all other variables equal, should withstand greater impact loading than prior art or
20 conventional bits. The more robust, stronger shirrtail 14, and particularly the shirrtail tip, is likely to last longer and assist in protecting the bearing seal or seals longer. The thicker shirrtail tip 14 afforded by the present invention also provides more wear protection and
25 provides sufficient support for a greater amount of shirrtail tip hard facing 30 on the outer surface of shirrtail 14.

Furthermore, the present bit can be formed so that a
30 greater portion of the shirrtail 14 has a reduced carbon content as compared to prior art shirrtails. Carbon

migration into the shirrtail, particularly the shirrtail tip, occurs during the carbonising/heat treatment process. Hard facing 30 welded to shirrtail tip also may provide a source for carbon that can carbonise the shirrtail tip.

5 Because carbon adds wear resistance but increases brittleness, a shirrtail that is not carbonised throughout its entire shirrtail tip area will withstand higher impact loads.

10 If desired, another advantage of the thicker shirrtail is that wear resistant inserts or other devices can be inlaid, or formed, into the outer surface of the legs thicker shirrtail and closer to the shirrtail tip, as compared to prior art shirrtails. Yet another potential
15 advantage of the thicker shirrtail is that it will, if desired, allow the placement of wear rings, such as wear ring 97 of Figure 5, or other mechanisms for the outer seal to engage. The use of such devices assists in preventing wear to the seal and leg sealing surface.

20 The use of a canted backface allows the placement of one or more outer, or secondary, bearing seals, if desired. Thus, in another aspect, the present invention includes a dual bearing seal configuration with a canted backface,
25 such as shown for example in Figure 1. Dual seal configurations and related technology are disclosed in U. S. Patent Application Serial Number 09/201,614 entitled "Dual-Seal Drill Bit Pressure Communication System" and filed on November 30, 1998 and U. S. Patent Application
30 Serial Number 08/982,081 entitled "Sealed Bearing Drill Bit With Dual Seal Configuration" and filed on December 1, 1997

(equivalent to GB-A-2331775), both having a common assignee as the present application and both of which are hereby incorporated by reference in their entireties.

5 This aspect of the present invention has various advantages. First, dual seals can be included in the bit without weakening the leg shirrtail. Further, in some sized bits, an outer or secondary seal of a non-canted design could not be included because of the limited area in
10 the bit. Another advantage is that the outer seal can be both axially and radially energised. For example, the seal 40 of Figures 1, 5, and 6 is partially radially and partially axially energised. Effectiveness of the secondary seal is enhanced by having a radially energised
15 component because radial energisation provides less contact pressure fluctuation of the sealing face caused by axial cone movement. Axial cone movement is inherent in roller cone bits. Similarly, by including an axial component in the direction of energisation, seal 40 can be made less
20 susceptible to relative lateral movement of the cone and leg.

 The construction, orientation, shape, configuration and location of the primary seal and the secondary seal 40
25 can take any suitable form. In Figures 2 and 5, for example, the seal 40 is an elongate, or bullet, O-ring seal having a fabric inlay 78 on an outer portion of the seal. Examples of fabric seals that are suitable for use in the present invention are disclosed and described in detail in
30 US-A-5842700. In Figures 5 and 6, the seals 40 are elongate O-ring seals having fabric inlays 44 across their

wear faces and partially along the adjacent sides. By way of further example only, seal 40 could be a dual-elastomer seal, and could include fabric, metal or other material components, could be a wiper or scraper seal, also referred to as an excluder, or a combination seal, such as a metal face or half-cat seal or a seal having any other form or configuration. A wiper or scraper seal is generally a non-positive seal that excludes debris from passing the seal and entering the bearing system.

10

Likewise, the canted seal of the present invention is useful, for example, as an outer seal with the canted backface of the present invention. This allows the use of a dual seal configuration without sacrificing bit integrity and with a secondary seal that is partially axially and partially radially energised. However, it should be understood that the canted seal is not limited to being used with the canted backface or any other aspect of this invention and may be used in other configurations.

20

It should further be understood that the present invention does not require that all or any particular combination of the above features be used together. They may each be used independently of the others, and with other features, such as a pressure communication system or device, wear rings or similar devices. For example, the canted backface, semi-encapsulated seal, and canted seal can each be used alone or in combination. Similarly, the canted seal and canted backface disclosed herein can be used in combination with either TCI or mill tooth bits, such as are known in the art. Likewise, various portions

30

of the present bit can be coated with one or more layers of
hardfacing material, or may have one or more wear resistant
inserts embedded therein. Surfaces where this may be
particularly desirable are typically the wear surfaces of
5 the bit, including the inner and/or outer surfaces of the
shirttail, and the cone backface.

It is further contemplated that one or more of seal 40
and any additional seals that are used between cone 20 and
10 leg 10 may be non-sealing members such as wiper rings or
the like, rather than elastomeric sealing members.

While various preferred embodiments of the invention
have been shown and described, modifications thereof can be
15 made by one skilled in the art without departing from the
scope of the present invention. The examples described
herein are exemplary only, and are not intended to limit
the scope of the invention. For example, the combinations
of features described herein, and the dimensions,
20 configuration, relative positioning and structure of the
components themselves, can each be modified in accordance
with known principles.

CLAIMS

1. An earth-boring bit for drilling a borehole, the bit comprising:
 - 5 a bit body, said body having at least one leg having a journal segment extending inwardly and downwardly therefrom, said leg including a shirttail, said shirttail having an outer surface and a leg backface; and,
at least one rolling cone cutter rotatably mounted on
10 said bit body for rotation about a bearing axis and having a generally conical outer surface and a cone backface, said cone backface defining an interface with said leg backface; wherein said leg backface is canted.
- 15 2. A bit according to claim 1, wherein said backface is canted at an angle of between 35 and 85 degrees with respect to the bearing axis.
3. A bit according to claim 1, wherein said backface is
20 canted at an angle of between 45 and 75 degrees with respect to the bearing axis.
4. A bit according to any of claims 1 to 3 wherein said cone backface is frustoconical.
- 25 5. A bit according to any of claims 1 to 4, comprising a seal located at the interface between the canted leg backface and the cone backface.
- 30 6. A bit according to any of claims 1 to 5, wherein said shirttail is recessed.

7. A bit according to claim 6 when dependent on claim 5, wherein at least a portion of the seal is located in the recess of the leg backface.

5

8. A bit according to any of claims 1 to 5, wherein said shirrtail is not recessed.

9. An earth-boring bit for drilling a borehole, the bit
10 comprising:

a bit body, said body having at least one leg having a journal segment extending inwardly and downwardly therefrom, said leg including a shirrtail, said shirrtail having an outer surface and a canted leg backface;

15 at least one rolling cone cutter rotatably mounted on said bit body and having a generally conical outer surface and a cone backface, said backface defining an interface with said shirrtail inner surface; and,

a seal positioned in said interface.

20

10. A bit according to claim 9, wherein said seal has an operating axis which is not normal to said canted backface.

11. A bit according to claim 9 or claim 10, wherein said
25 shirrtail is recessed.

12. A bit according to claim 9 or claim 10, wherein said shirrtail is not recessed.

30 13. A bit according to any of claims 9 to 12, wherein said seal is a non-positive seal.

14. An earth-boring bit for drilling a borehole, the bit comprising:

5 a bit body, said body having at least one leg having a journal segment extending inwardly and downwardly therefrom, said leg including a shirttail, said shirttail having an outer surface and a leg backface;

10 at least one rolling cone cutter rotatably mounted on said bit body and having a bearing axis, a generally conical outer surface, and a backface, said backface defining an interface with said shirttail inner surface; and,

15 a canted seal positioned in said interface, said seal having an operating axis that is canted with respect to said bearing axis.

15. A bit according to claim 14, wherein said seal has an elongate cross-section and includes a fabric inlay on its wear face.

20

16. A bit according to claim 14 or claim 15, wherein said seal is received in a groove in said cone and bears on said leg backface.

25 17. A bit according to claim 16, wherein said leg backface includes a wear resistant insert.

18. A bit according to claim 14 or claim 15, wherein said seal is received partially in a groove in said cone and
30 partially in a cut-out in said leg backface.

19. A bit according to claim 18, wherein said cut-out includes a wear resistant insert.

20. A bit according to any of claims 14 to 19, wherein
5 said leg backface is canted.

21. A bit according to any of claims 14 to 19, wherein said leg backface is normal to said bearing axis.

10 22. A bit according to any of claims 14 to 21, wherein said shirrtail is recessed.

23. A bit according to any of claims 14 to 21, wherein said shirrtail is not recessed.

15

24. A bit according to any of claims 14 to 23, wherein said seal operating axis is canted at an angle of between
15 and 45 degrees with respect to said bearing axis.

20 25. A bit according to any of claims 14 to 23, wherein said seal operating axis is canted at an angle of between
70 and 85 degrees with respect to said bearing axis.

26. A bit according to any of claims 14 to 25, wherein
25 said seal is a non-positive seal.

27. A bit according to any of claims 14 to 26, further including a second seal positioned between said journal segment and said cone.

30

28. A bit according to claim 27, wherein said second seal is canted.

29. A drill bit, substantially in accordance with any of
5 the examples as hereinbefore described with reference to
and as illustrated by Figures 1, 2, 4, 5 and 6 of the
accompanying drawings.



INVESTOR IN PEOPLE

Application No: GB 0002397.8
Claims searched: 1-13

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Patents Act 1977 Search Report under Section 17

Databases searched:

UK Patent Office collections, including GB, EP, WO & US patent specifications, in:

UK Cl (Ed.R): E1F: FFG, FFU, FFE

Int Cl (Ed.7): E21B:

Other: Online: JAPIO, WPI, EPODOC

Documents considered to be relevant:

Category	Identity of document and relevant passage	Relevant to claims
X	GB2293841 A SMITH INTERNATIONALINC (See fig. 2)	1,4,5,8,9,10,12
X	US4448268 FULLER (See fig. 4)	1,4-7,9,11
X	WO80/02037 SANDVIK (See fig. 1)	1,4,5,8,9,10,12,13

X	Document indicating lack of novelty or inventive step	A	Document indicating technological background and/or state of the art.
Y	Document indicating lack of inventive step if combined with one or more other documents of same category.	P	Document published on or after the declared priority date but before the filing date of this invention.
&	Member of the same patent family	E	Patent document published on or after, but with priority date earlier than, the filing date of this application.